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Design tables for folded plates

M. G. Tamhankar



These tables are offered as a basis for making a choice of three commonly employed shapes of folded plates and fixing dimensions for a preliminary design. The logic behind the tables, explained in detail in the article, is that once the stress resultants for a particular span are known, they can be found for another span.

Folded plates are now extensively used to roof large column-free areas and have been found ideally suitable for industrial structures. Architects and engineers in India have lately become interested in this form of construction, various papers published by the Central Building Research Institute having contributed to some extent in popularising it.^{1,2,3,4,5} This paper is another step in this direction. It aims at providing ready-made design tables of stress resultants for a large number of folded plates. Commonly employed shapes of folded plates have been chosen and transverse moments and longitudinal stresses for different spans have been presented.

Method of analysis

The detailed study of the various methods of folded plate analysis has revealed that the Simpson method⁶ is more flexible than the Whitney method⁷ in arriving at the stress resultants for various spans with a constant cross-section. The procedure which forms the basis for the design tables is enumerated in the following paragraphs. Readers are referred to the original papers of Simpson and Whitney for a better understanding of the procedure.

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the 'rotation' solutions

The necessary changes to be incorporated in the Simpson and Whitney methods for the changes in span are enumerated below. The Simpson method is explained in somewhat greater detail than the Whitney method as the former is recommended for preparing similar tables.

Simpson's method

- 1. The external load is divided into components parallel and perpendicular to the plate on which it acts. A strip of unit width is then analysed as a continuous beam in the transverse direction, the ridges being assumed unyielding. Thus, transverse moments m^0_k and plate loads R^0_k are obtained. This step is referred to as the 'no rotation' solution.
- 2. From the plate loads R^{o}_{k} , longitudinal stresses $\frac{R^{o}_{k}L^{2}}{8 Z_{k}}$ and subsequent 'distributed' stresses $A_{k}L^{2}$ are obtained.
- 3. To account for the transverse relative displacements of the joints each plate is rotated, in turn, arbitrarily, and transverse moments $M_{k'}$ and longitudinal stresses $A_{k'}$ L^2 for every case of 'rotation' are obtained. These calculations are referred to as the 'rotation' solutions.

NOTATION

- L = span of folded plate between diaphragms Sok tk = thickness of plate kwk = width of plate k δ_{k}' γ_k = angle between two plates at k measured from the inferior plate in the clockwise Δ_k direction towards the superior plate Z_k ψ_k = section modulus of plate k= load in plane of plate k for the 'no rota- ψ^{o}_{k} tion' solution Kmok = transverse moments at joint k for the 'no rotation' solution M_{k}' = transverse moment at joint k for one of
 - k = in-plane deflection of plate k for the 'no rotation' solution
 - $\delta_{k'}$ = in-plane deflection of plate k for one of the 'rotation' solutions
 - Δ_k = final in-plane deflection of plate k
 - ψ_k = final rotation of plate k
 - ψ^{o}_{k} = arbitrary rotation of plate k
 - K = arbitrary constant which is the ratio of the final rotation to the arbitrary rotation of the plate.
 - A_k , $A_{k'}$, $B_{k'}$, B_k , etc = constants depending upon the cross-sectional properties

- 4. From the stresses at the top and bottom of each plate, in-plane deflections of the plates are obtained for the 'no rotation' solution $[\delta^o_k = B_k L^4]$ and the 'rotation' solution $[\delta_{k'} = B_{k'} L^4]$.
- 5. Final in-plane deflection of the plate k can be expressed as follows:

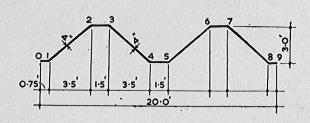
$$\Delta_{k} = \delta^{o}_{k} + K_{1} \delta_{k'} + K_{2} \delta_{k''} + K_{3} \delta_{k'''} + \dots$$

$$= B_{k} L^{4} + K_{1} B_{k'} L^{4} + K_{2} B_{k''} L^{4} + K_{3} B_{k'''} L^{4} + \dots$$

$$(1)$$

Here, K_1 , K_2 , etc are the arbitrary constants for the plates 1, 2, etc and are equal to the ratio of the actual plate rotation to the arbitrary rotation, *i.e.*, $\psi_k = K \psi^o_k$.

 Knowing the in-plane deflections, rotations of the plates ψ_k are obtained from the formula based on the deformed geometry of the cross-section.



$$\psi_{k} = \frac{1}{w_{k}} \left[\Delta_{k} \left(\cot \Upsilon_{k} + \cot \Upsilon_{k-1} \right) - \frac{\Delta_{k+1}}{\sin \Upsilon_{k}} - \frac{\Delta_{k-1}}{\sin \Upsilon_{k-1}} \right]$$

$$= \frac{L^4}{w_k} \left[\text{ Terms in } k_1, k_2, \text{ etc and } \right] \qquad (3)$$

7. But $\psi_k = K \psi^0_k$ and the arbitrary rotation is proportional to $\frac{w_k}{t^3_k}$, hence, taking the factor $\frac{L^4}{w_k}$ of the equation (3) to the left-hand side, we get

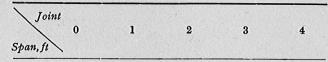
8. Simultaneous equations resulting from equation (4) are solved for K_1, K_2, K_3 , etc. Knowing these arbitrary

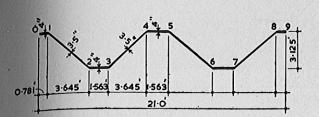
Live load: 10 lb/ft2 of surface area

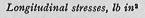
					our race are
Jo Span, fi	oint 0	1	2	3	4
		Longitudina	al stresses, ll	/in ²	
60	+ 670.00	+ 800.90	- 762·10	- 755·10	+ 730.90
55	+ 563.00	+ 672 90	- 640 • 40	- 634.60	+ 614.10
50	+ 480.18	+ 573.97	- 546.15	- 541 - 24	+ 523.82
		Transverse	moments, lb	ft ft	
60	_	+ 16.88	+ 923 · 80	+ 898 • 60	— 161·20
55	.—	+ 16.88	+ 923.05	+ 892 • 62	- 197.37
50	-	+ 16.88	+ 920.00	+ 883 • 00	- 240.00
AND DESCRIPTION OF THE PARTY OF					

Live load: 15 lb/ft2 of surface area

							3 10/16-01	surface area
Span,	ft ft	0	1		2		3	4
			Longia	udina	ıl stress	es, lb	/in ²	
65	-	1118 • 68	- 822	·87 ·	+ 874 •	75 -	+ 904.00	- 863.99
60	-	946.82	- 705	•35	+ 747 ·	92 -	+ 769 · 37	- 734·51
55	-	784.00	- 598	·61 ·	+ 632.	25 -	+ 643.39	- 615 ·38
50	_	638 • 36	- 499	·61 -	+ 525.	65 -	+ 529 · 18	— 507·09
			Trans	ierse 1	noment	s, lb j	ft ft	
65		_	+ 32	•50 -	+ 900 •	85 -	+ 847.98	_ 256.75
60		_	+ 32	-50 -	+ 900 •	23 -	+ 845.30	- 266.22
55		-	+ 32	-50 -	+ 898	91 -	+ 841.93	- 274.90
50			+ 32	50	+ 897	60 -	⊢ 838 • 66	- 280.96





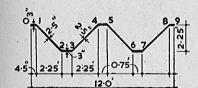


60	· -	+	19.84	+ 985.58	+ 953.07	- 271 · 23
55	_	+	19.84	+ 982.59	+ 948.20	- 284.06
50	_	+	19.84	+ 979.59	+ 943.35	- 296.88
45	_	+	19.84	+ 976.59	+ 938.50	- 309.70

Live load: 10 lb/ft2 of surface area

Joint					
	0	1	2	3	4
Span, ft					

Longitudinal stresses, lb/in2



Live load: 10 lb/ft2 of surface area

+4.25 + 204.00 + 199.10 - 38.23

Joint						
30	0	1	2	٥	3	4
Span, ft						

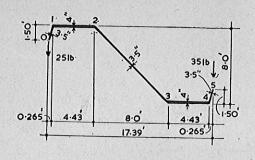
40

Longitudinal stresses, lb/in2

60	- 839·05 -	1067 • 51 +	1007 · 44	+ 988.75	- 947 ·96
55	- 705·04 -	897.01 +	846.53	+ 830.83	- 796·55
50	- 582·68 -	741.33 +	699 • 61	+ 686.63	- 658·30
45	- 471·96 -	600.48 +	566 · 68	+ 556.17	- 533 ·23
	Tra	nsverse moi	nents, lb	ftlft	

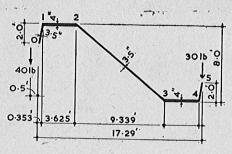
60	-	+5.79 + 284.52	+ 279.38	- 38.08
55	-	+5.79 + 284.21	+ 278 · 11	- 45.94
50	-, 0	+5.79 + 283.90	+ 276:84	- 53.80
AE		⊥ 5.70 ± 283.59	⊥ 275.57	_ 61.66

Northlight folded plates



Live load: 15 lb/ft2 of surface are

				VIII -	- To Te of surface ar
0	1	2	3	4	5
1997	Long	gitudinal stresses, lb/	in ²		
$+\ 1467 \cdot 80$	+ 460.96	- 973 · 67	+ 980.49	- 472·02	- 1492.60
+ 1061.16	$+\ 301 \cdot 71$	- 663 · 08	+ 672.50	- 308.00	- 1081.36
+ 893.45	+ 229.48	- 534.60	+ 538.00	- 233.45	- 912.22
+ 705.94	+ 181.32	- 422.40	+ 425.09	- 184 · 45	- 720.77
	7	ransverse moments, l	b ft ft		
<u> </u>	+ 18.45	+ 682 · 19	+ 728.52	+ 21.09	
÷	+ 18.45	+ 657.83	+ 702.42	+ 21.09	_
	+ 18.45	+ 645.65	+ 689.42	+ 21.09	<u> </u>
<u> </u>	+ 18.45	+ 633.47	+ 676.42	+ 21.09	_
	+ 1467·80 + 1061·16 + 893·45 + 705·94	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	Longitudinal stresses, lb/ + 1467·80	$Longitudinal stresses, lb/in^{2}$ $+ 1467 \cdot 80 + 460 \cdot 96 - 973 \cdot 67 + 980 \cdot 49$ $+ 1061 \cdot 16 + 301 \cdot 71 - 663 \cdot 08 + 672 \cdot 50$ $+ 893 \cdot 45 + 229 \cdot 48 - 534 \cdot 60 + 538 \cdot 00$ $+ 705 \cdot 94 + 181 \cdot 32 - 422 \cdot 40 + 425 \cdot 09$ $Transverse moments, lb ft/ft$ $- + 18 \cdot 45 + 682 \cdot 19 + 728 \cdot 52$ $- + 18 \cdot 45 + 657 \cdot 83 + 702 \cdot 42$ $- + 18 \cdot 45 + 645 \cdot 65 + 689 \cdot 42$	$Longitudinal stresses, lb/in^{2}$ $+ 1467 \cdot 80 + 460 \cdot 96 - 973 \cdot 67 + 980 \cdot 49 - 472 \cdot 02$ $+ 1061 \cdot 16 + 301 \cdot 71 - 663 \cdot 08 + 672 \cdot 50 - 308 \cdot 00$ $+ 893 \cdot 45 + 229 \cdot 48 - 534 \cdot 60 + 538 \cdot 00 - 233 \cdot 45$ $+ 705 \cdot 94 + 181 \cdot 32 - 422 \cdot 40 + 425 \cdot 09 - 184 \cdot 45$ $Transverse moments, lb ft/ft$ $- + 18 \cdot 45 + 682 \cdot 19 + 728 \cdot 52 + 21 \cdot 09$ $- + 18 \cdot 45 + 657 \cdot 83 + 702 \cdot 42 + 21 \cdot 09$ $- + 18 \cdot 45 + 645 \cdot 65 + 689 \cdot 42 + 21 \cdot 09$



					Live load: 15	lb/ft2 of surface area
Joint Span, ft	0_	1	2	3	4	. 5
		Lon	ngitudinal stresses, l	lb in²		
60	+ 1640.00	+320.00	− 987·00	+ 984.50	- 321.80	- 1610.00
55	+ 1321 • 52	+ 301 · 14	- 844.45	+ 843.48	$-305 \cdot 04$	- 1299 • 20
45	+ 846.81	+ 223.17	- 575.40	+ 575.51	- 227.38	- 834.86
40	. + 669.08	+ 176.33	- 454·64	+ 454.72	- 179·66	- 659.64
		T_1	ransverse moments, l	lb fi fi		
60	— ·	+ 55.10	+ 242.10	+ 187.50	+ 31.60	
55	-	+ 55.10	+ 250.59	+ 196.44	+ 31.60	_
45		+ 55.10	+ 279 · 23	+ 227.48	+ 31.60	_
40		+ 55.10	+ 301.79	+ 252.73	+ 31.60	_

constants, final transverse moments at midspan can be obtained as follows:

$$M_k = m^o_k + \Sigma K M_k' \qquad (5)$$

The final longitudinal stresses at midspan can be obtained as

$$A_k L^2 + \sum K A_k' L^2$$

$$= L^2 \left[A_k + \sum K A_k' \right] \qquad (6)$$

Equations (5) and (6) reveal that for a given crosssection, the transverse moments depend upon the values of the arbitrary constants K, whereas the longitudinal stresses depend upon the arbitrary constants K as well as the span of the folded plate.

Thus, if a problem is solved for a span L and for a certain cross-section and then, if the stress-resultants are required for some other span L, the only change to be incorporated is in equation (4) of step 7, all other steps remaining unchanged, i.e., the left-hand side of

equation (4) must be multiplied by a factor $\left[\frac{L^4}{L'^4}\right]$

The Whitney method

The only revision necessary in the problem when it is solved by the Whitney method is in the final set of simultaneous equations. These equations are based on the condition that the total change in the angle at any joint is zero. The change in angle results from

the rotations of the plates at the joints caused due to 'slab action' of the external loads

the rotations of the plates at the joints caused due to the transverse moments

the angle change caused due to the in-plane deflections of the plates.

Of the three quantities, the first two remain constant irrespective of the span. Thus, the only change occurs in the third quantity. Unlike in the Simpson method, the unknowns in the final set of equations are transverse moments. As such, they can be obtained very easily for any span. But the calculations for longitudinal stresses involve evaluation of intermediate quantities like plate moments and longitudinal shears.

After solving a large number of folded plates, it has been observed that the term on the left-hand side of equation (4) in the Simpson method is either very small or fairly large compared to its corresponding term on the right-hand side. As such, when a problem is solved taking into account the term on the left-hand side, any

subsequent multiplication of that term with $\left[\frac{L^4}{L'^4}\right]$ would

make very little change in the magnitude, if the quantity is initially large or the entire quantity on the left-hand side becomes negligible, if it is initially small. Thus one set of K values will hold good for a certain range of spans, *i.e.*, the transverse moments in that range remain constant irrespective of the span and the longitudinal stresses are directly proportional to L^2 for all practical purposes. In fact, these are the conclusions drawn in the simple beam theory. It is also well known that the beam theory has limited application to folded plates. There has, therefore, to be a limitation on the

value of the term $\left[\frac{L'^4-L^4}{L'^4}\right]$ so that the above conclusion may remain valid.

The entire problem thus centres on the sensitivity of the left-hand side quantity of equation (4) for the Simpson method, and the relative magnitude of the angle change due to the plate deflections compared to the first two quantities in the Whitney method. In one of the problems, it was noticed that even the total omission of the quantity on the left-hand side in equation (4) made very little difference in the final stress resultants. However, the transverse moments are invariably found to be more sensitive to the K values than the longitudinal stresses.

In the design tables, for certain shapes, simultaneous equations were solved for every span. In certain cases the transverse moments were obtained for every span but the longitudinal stresses were deducted proportional to L^2 . However, the procedure discussed in this paper enables us to work out stress resultants for any range of spans by adopting the following steps:

- 1. A problem is solved by the Simpson method for a particular span, say 50 ft.
- 2. This solution enables us to find out stress resultants within a range of 45 ft to 55 ft. The longitudinal stresses are assumed proportional to L^2 and the transverse moments constant within the range of \pm 5 ft.
- 3. The final set of simultaneous equations arising from equation (4) are solved again for a span, say, 65 ft in place of 50 ft. This amounts to a change in the left-hand side term of the order of $\left[\frac{L'^4-L^4}{L'^4}\right]$. This solution furnishes stress resultants for a range of 60 ft to 70 ft.

Thus, the variation of the stress resultants over a wide range of 45 ft to 70 ft is known sufficiently accurately to proceed with the preliminary design or to choose the dimensions of the folded plate.

All these explanations are given here to indicate the logic employed in the design tables. As far as the designer is concerned, he may pick any one of the shapes and a suitable corresponding span from the tables.

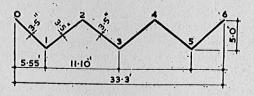
Design tables

The tables given herein pertain to three shapes which are most frequently used. Structural depths have been chosen so that the final stresses in the given range of spans are somewhere around the permissible stresses. A live load of 15 lb/ft² of surface area is assumed, unless specified in the problem. The span variation is limited to the range of 40 ft to 70 ft as prestressing would be generally necessary for spans exceeding 70 ft, and folded plates for less than 40 ft are generally uneconomical.

Conclusions

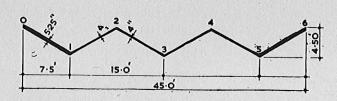
1. The design tables offer a basis for comparing economical proportioning of cross-sections for a given span. In fact, this is the problem that faces a designer when he

(· · · ·		-110	1044. 10 10/10	of surface are
Joint Span, ft	0	1	2	3
	Lo	mgitudinal stre	sses, lb/in²	
65	- 870·55	+ 878.83	+ 826.05	+ 764.98
60	- 741·77	+ 748.82	- 703·85	+ 651.82
55	- 623·29	+ 629.22	- 591 • 43	+ 547.71
	Tra	nsverse momen	ts, lb ft/ft	
65	_	+ 1120-43	- 68.40	+ 317.90
60		+ 1120-43	- 77.00	+ 342 - 20
55	_	+ 1120 · 43	- 85.61	+ 366.50



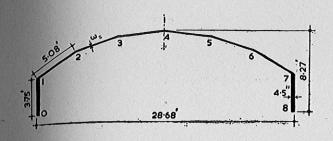
Live load: 15 lb/ft2 of surface area

Span,	oint 0	1	2	3
		Longitudinal stre	sses, lb/in²	
6	-928.0	$0 + 973 \cdot 00$	- 935.00	+ 842.00
6	0 - 792.0	0 + 829.00	− 797·00	+ 715.00
5	665.0	0 + 697.00	- 670.00	+ 600.00
		Transverse momen	its, lb ft/ft	
68	-	+ 2644 • 60	- 339.30	+ 832 • 20
60	· –	+ 2644.60	- 328.30	+ 793 · 10
58	–	+ 2644 · 60	- 320 • 20	+ 761 · 90



Live load: 15 lb/ft2 of surface area

Live load : 15 lb/it- of surface a								
Joint Span, ft	. 0	1	2	3				
	Lo	ngitudinal stre	sses, lb/in²		100			
70	- 834.96	+ 874 · 32	- 839 • 49	+ 754 · 15				
65	- 719·94	+ 753.88	— 723·85	+ 650 · 26				
60	- 613 • 44	+ 642.36	- 616 · 77	+ 554.07				
	Tre	ansverse momer	it lb ft ft					
70	_	+ 2566 · 15	- 299.68	+ 743.77				
65	_	+ 2566 • 15	- 294 - 57	+ 724.56				
60	_	+ 2566.15	- 290.51	+ 709.30				



Jo Span,	oint 0	. 1	2	3	4
		Longitudin	al stresses, i	b/in ²	1
70	+ 859.33	+ 188.33	- 97.14	- 387 · 48	- 400·75
65	+ 742.47	+ 164 · 63	- 92.09	- 333.34	- 336.59
60	+ 632.64	+ 140.28	- 78.47	- 284.03	- 286.80
		Transverse	moments, l	b ft	
70	 >	_	- 212·80	+ 206.35	+ 436.83
65	-	<u> </u>	— 215·71	+ 198.92	+ 426.99
60	-	_	- 218.62	+ 191 · 49	+ 417.15

is fixing the preliminary dimensions. Since the quantity of mild steel in any proposal depends upon the values of longitudinal stresses and transverse moments, the tables present very useful information for purposes of preliminary design.

2. On comparing the various shapes for a constant span, it is observed that the trough shape is more efficient than the V. The efficient structural depth for a trough type folded plate varies between 1/15 to 1/18 of the span, whereas for the V-shape, a height of 1/10 to 1/12 of the span is necessary. For northlight folded plates the structural depth (in the real sense) being inclined, a vertical depth of $\frac{1}{6}$ to $\frac{1}{8}$ of the span is required.

3. Transverse moments assume a proportionately high value at joints 2 and 3 for a trough shape. For a constant trough type of cross-section, as the span is reduced the transverse moments at joints 2 and 3 increase, whereas the transverse moment at joint 4 decreases. The largest value of the transverse moments for a V-shape is the free cantilever moment at the joint 1.

4. Stresses of very high order are observed at the free edges of the northlight folded plates. However, the stress level in the interior is fairly low.

Acknowledgments

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